

FREQUENCY DOMAIN ANALYSIS

Email: hasansaeedcontrol@gmail.com

URL: [http://shasansaeed.yolasite.com/
hasansaeed.872726549.wordpress.com](http://shasansaeed.yolasite.com/hasansaeed.872726549.wordpress.com)

BOOKS

1. AUTOMATIC CONTROL SYSTEM *KUO & GOLNARAGHI*
2. CONTROL SYSTEM *ANAND KUMAR*
3. AUTOMATIC CONTROL SYSTEM *S.HASAN SAEED*

FREQUENCY DOMAIN ANALYSIS

FREQUENCY RESPONSE:

- The magnitude and phase relationship between sinusoidal input and steady state output of a system is known as frequency response.
- It is independent of the amplitude and phase of the input signal.
- If input signal is $r(t) = X \sin \omega t$ then output can be written as

$$c(t) = Y \sin(\omega t + \theta)$$

ADVANTAGES OF FREQUENCY RESPONSE:

- Frequency response tests are simple to perform.
- Transfer function can be obtained from frequency response of the system.
- Frequency response methods can be used to find the absolute as well as relative stability of the system.
- The apparatus required for obtaining frequency response are simple and easy to use.
- The effect of noise disturbance and parameter variations are relatively easy to visualize and assess through frequency response.

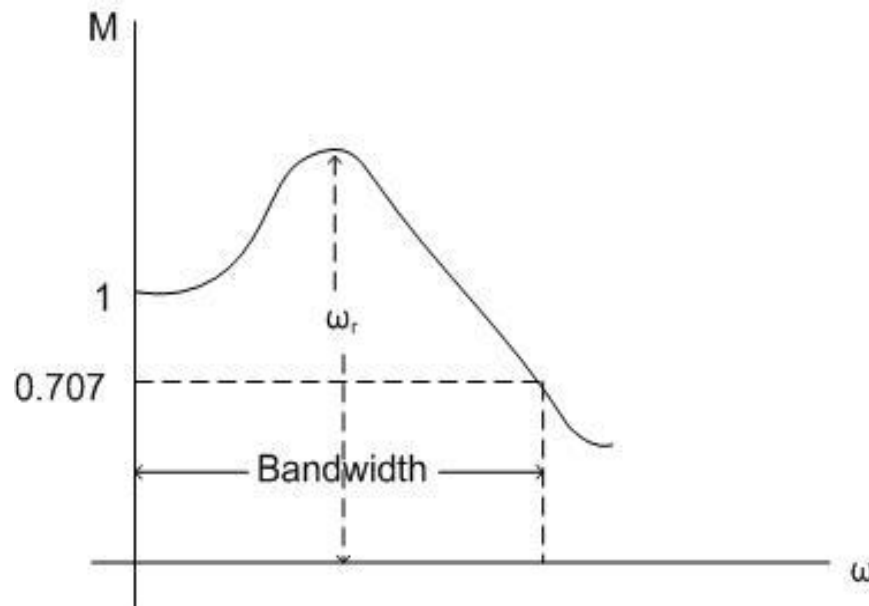
DISADVANTAGES OF FREQUENCY RESPONSE

- Obtaining frequency response practically is fairly time consuming.
- These methods are applied to linear systems.
- Frequency response test cannot be performed on non-interruptable systems.
- For high time constant, frequency response method is not convenient.

FREQUENCY DOMAIN SPECIFICATIONS

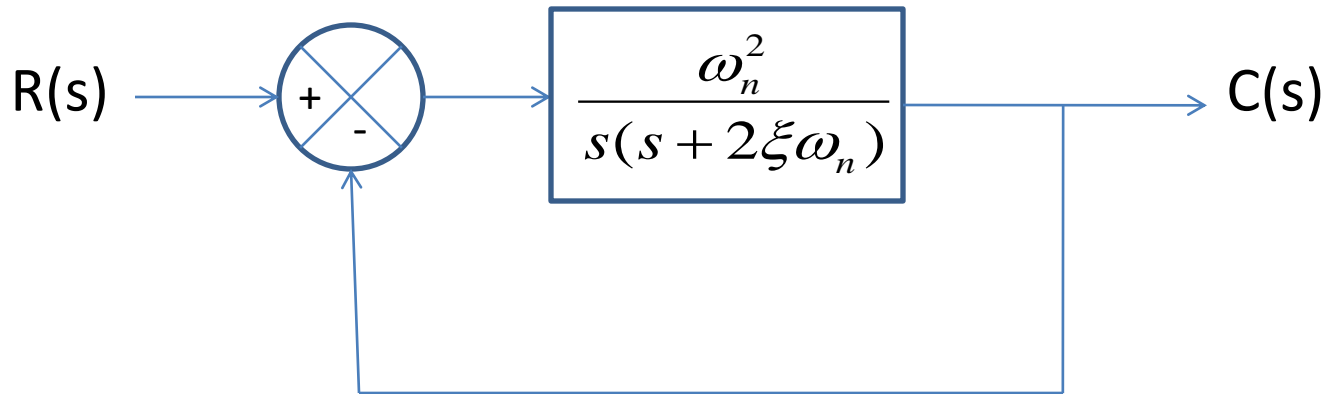
- **Resonant peak(M_r):** The maximum value of magnitude is known as resonant peak. The magnitude of resonant peak gives the information about the relative stability.
- **Resonant Frequency (ω_r):** The frequency at which magnitude has maximum frequency is known as resonant frequency.
- **Bandwidth:** Bandwidth is defined as the range of frequencies in which the magnitude of closed loop does not drop -3db. Or bandwidth is defined as the band of frequencies lying between -3db points.

- **Cut-off Frequency:** The frequency at which the magnitude is 3db below its zero frequency value is called cut-off frequency (ω_b).
- **Cut-off Rate:** The cut-off rate is the slope of the log magnitude curve near the cut off frequency.



Correlation between Time and Frequency Response

Consider the second order system



$$\frac{C(s)}{R(s)} = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2} \text{ --- (1)}$$

Put $s=j\omega$ in equation (1)

$$\frac{C(j\omega)}{R(j\omega)} = \frac{\omega_n^2}{(j\omega)^2 + 2\xi\omega_n(j\omega) + \omega_n^2} = \frac{1}{\left(1 - \frac{\omega^2}{\omega_n^2}\right) + j2\xi \frac{\omega}{\omega_n}}$$

Let $\frac{\omega}{\omega_n} = u$

$$\frac{C(j\omega)}{R(j\omega)} = \frac{1}{(1 - u^2) + j2\xi u}$$

$$M(j\omega) = \frac{C(j\omega)}{R(j\omega)} = \frac{1}{\sqrt{(1 - u^2)^2 + (2\xi u)^2}} \text{-----} (2)$$

$$\theta = -\tan^{-1} \frac{2\xi u}{1-u^2} \text{-----}(3)$$

The steady state output for a sinusoidal input of unit magnitude and variable frequency

$$C(t) = \frac{1}{\sqrt{(1-u^2)^2 + (2\xi u)^2}} \sin \left[\omega t - \tan^{-1} \frac{2\xi u}{1-u^2} \right] \text{-----}(4)$$

From equation (2) & (3)

u	M	θ
0	1	0^0
1	$\frac{1}{2\xi}$	$-\frac{\pi}{2}$
∞	0	$-\pi$

The frequency at which 'M' has maximum value is known as resonant frequency ω_r

Put $u_r = \frac{\omega_r}{\omega_n}$

Where u_r is normalized resonant frequency

Differentiate equation (2) w.r.t u and put $u = u_r$

We get $4u_r^3 - 4u_r + 8u_r\xi^2 = 0$

$$u_r = \sqrt{1 - 2\xi^2}$$

$$u_r = \frac{\omega_r}{\omega_n}$$

$$\omega_r = \omega_n \sqrt{1 - 2\xi^2} \text{ --- (5)}$$

from equation (3)

$$\theta = -\tan^{-1} \frac{2\xi u_r}{1 - u^2}$$

$$\therefore \theta = -\tan^{-1} \frac{\sqrt{1 - 2\xi^2}}{\xi}$$

For maximum value of magnitude put
in equation (2)

$$u_r = \sqrt{1 - 2\xi^2}$$

$$\therefore M_r = \frac{1}{2\xi\sqrt{1-\xi^2}} \text{-----} (6)$$

For step response of second order system, maximum overshoot is given by

$$M_p = e^{-\frac{\pi\xi}{\sqrt{1-\xi^2}}} \text{-----} (7)$$

Damped frequency of oscillation is given by

$$\omega_d = \omega_n \sqrt{1 - \xi^2} \text{-----} (8)$$

From equation (7) & (8)

$$\omega_r = \omega_d \left[\frac{\sqrt{1 - 2\xi^2}}{\sqrt{1 - \xi^2}} \right] \text{-----} (9)$$

From above equations it is clear that if the value of ξ is small, the damped frequency ω_d and resonant frequency are nearly same. Hence for large value of ω_r the time response is faster.

As ξ increases, M_p decreases and gets vanished when $\xi = 1$. After that, system does not produce any overshoot. While in frequency domain M_r will Vanish if

$$\sqrt{1 - 2\xi^2} = 0$$

$$\therefore \xi = 0.707$$

and in such case system will not exhibit resonant peak.

When ξ is very small i.e less than 0.4, both M_p and M_r will be very large and are not desirable. So ξ is generally designed to be between $0.4 < \xi < 0.707$

BANDWIDTH: The bandwidth is defined as the range of frequencies over which M is equal or greater than 0.707

From equation (2)

$$M = \frac{1}{\sqrt{(1-u_b^2)^2 + (2\xi u)^2}} = \frac{1}{\sqrt{2}}$$

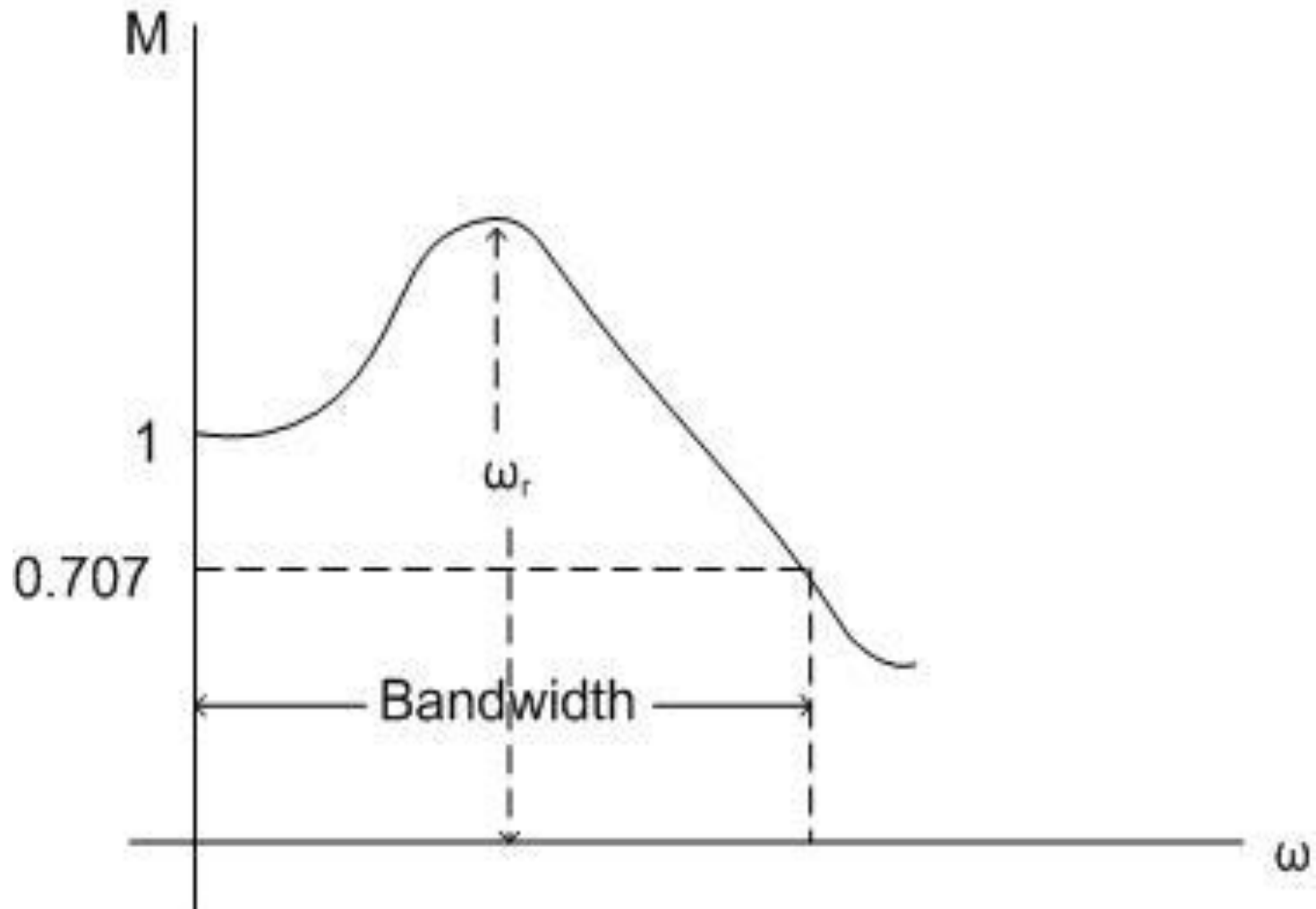
$$u_b^4 - 2u_b^2(1 - 2\xi^2) - 1 = 0$$

$$u_b = \sqrt{1 - 2\xi^2} + \sqrt{2 - 4\xi^2 + 4\xi^4}$$

$$u_b = \frac{\omega_b}{\omega_n}$$

$$\omega_b = \omega_n \sqrt{1 - 2\xi^2} + \sqrt{2 - 4\xi^2 + 4\xi^4}$$

M Vs ω



Example: The forward path transfer function of a unity feedback control system is

$$G(s) = \frac{100}{s(s + 6.54)}$$

Find the resonance peak, resonant frequency and bandwidth of the closed loop system.

Solution: Given that

$$G(s) = \frac{100}{s(s + 6.54)}$$

$$H(s) = 1$$

$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)H(s)}$$

$$\frac{C(s)}{R(s)} = \frac{100}{s^2 + 6.54s + 100}$$

Compare with $\frac{C(s)}{R(s)} = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$

$$\omega_n^2 = 100$$

$$\omega_n = 10 \text{ rad / sec}$$

$$2\xi\omega_n = 6.54$$

$$2\xi * 10 = 6.54$$

$$\xi = 0.327$$

$$\omega_r = \omega_n \sqrt{1 - 2\xi^2}$$

$$\omega_r = 10 \sqrt{1 - 2(0.327)^2} = 8.86 \text{ rad / sec}$$

$$M_r = \frac{1}{2\xi\sqrt{1-\xi^2}}$$

$$M_r = \frac{1}{2(0.327)\sqrt{1-(0.327)^2}}$$

$$M_r = 1.618$$

$$\omega_b = \omega_n \sqrt{1 - 2\xi^2 + (2 - 4\xi^2 + 4\xi^4)^{1/2}}$$

$$\omega_b = 10\sqrt{1 - 2(0.327)^2 + (2 - 4 * 0.327^2 + 4 * 0.327^4)^{1/2}}$$

$$\omega_b = 14.34 \text{ rad / sec}$$

CONSTANT MAGNITUDE CIRCLE (M-CIRCLE)

Let $G(j\omega) = x + jy$

Then from $\frac{C(j\omega)}{R(j\omega)} = \frac{G(j\omega)}{1 + G(j\omega)}$

$$M = M(j\omega) = \frac{|C(j\omega)|}{|R(j\omega)|} = \left| \frac{G(j\omega)}{1 + G(j\omega)} \right| = \frac{|x + jy|}{|1 + x + jy|} = \frac{\sqrt{x^2 + y^2}}{\sqrt{(1+x)^2 + y^2}} \text{ --- (1)}$$

$$M^2 = \frac{x^2 + y^2}{(1+x)^2 + y^2}$$

$$M^2[(1+x)^2 + y^2] = x^2 + y^2$$

$$M^2[1 + 2x + x^2 + y^2] = x^2 + y^2$$

$$x^2(1-M^2) + y^2(1-M^2) - 2xM^2 = M^2$$

Divide both side by $(1-M^2)$

We get $x^2 - y^2 - 2x\frac{M^2}{1-M^2} = \frac{M^2}{1-M^2}$

$\left[\frac{M^2}{1-M^2}\right]^2$ Add both sides

We get

$$\left[x - \frac{M^2}{1-M^2}\right]^2 + (y-0)^2 = \frac{M^2}{(1-M^2)^2} \text{-----} (2)$$

Equation (1) is the equation of the circle with centre $\left[\frac{M^2}{1-M^2}, 0 \right]$ and radius $\left[\frac{M}{1-M^2} \right]$

If $M=1$ then equation (1) becomes

$$(1+x)^2 + y^2 = x^2 + y^2$$

$$x = -\frac{1}{2}$$

This is the equation of the straight line parallel to the y-axis and passing through $(-1/2, 0)$ in $G(j\omega)$ plane.

The constant M locii for different values of M is shown in fig, it is clear that

- (a) The locii are symmetrical with respect to $M=1$
- (b) The M -circle for $M>1$ are on left side of the line $M=1$ and for $M<1$ the circles are on right side of the line $M=1$
- (c) The intersection of $G(j\omega)$ plot (Nyquist plot) and constant M locii gives the value of magnitude the M -circle which is tangent to the $G(j\omega)$ plot give the value of resonance peak and resonant frequency.

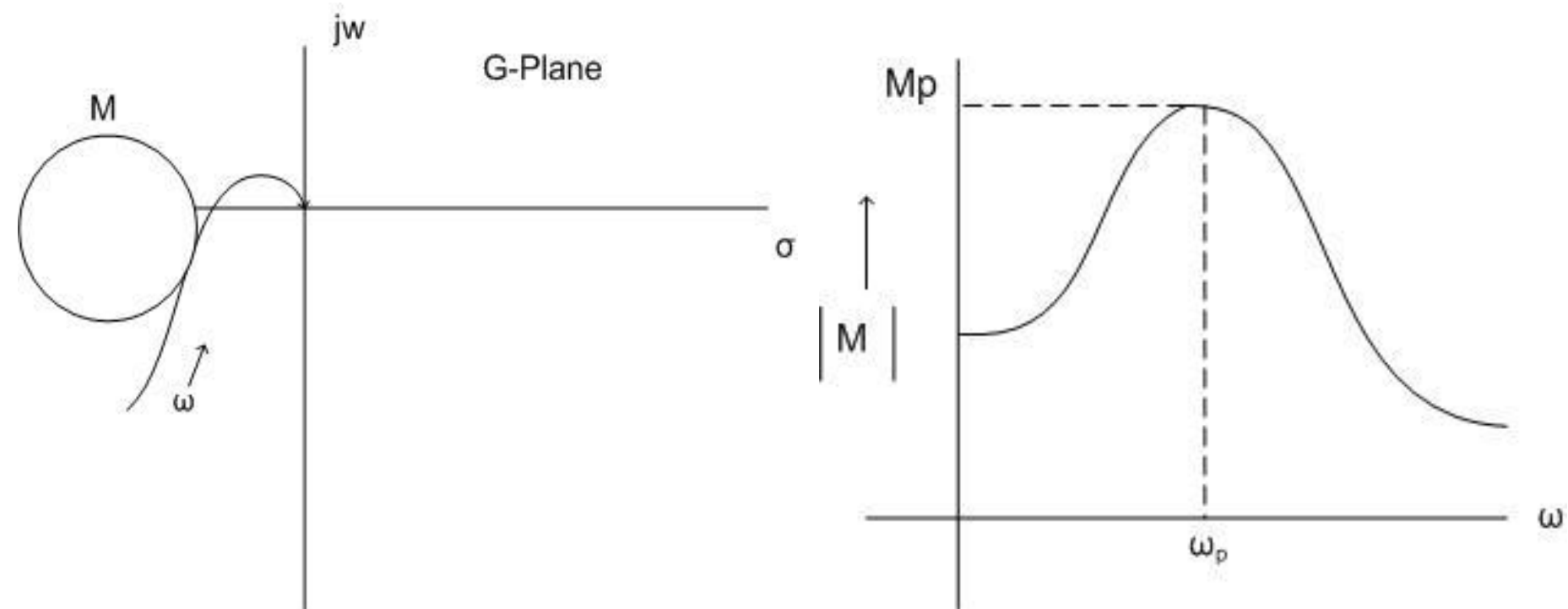
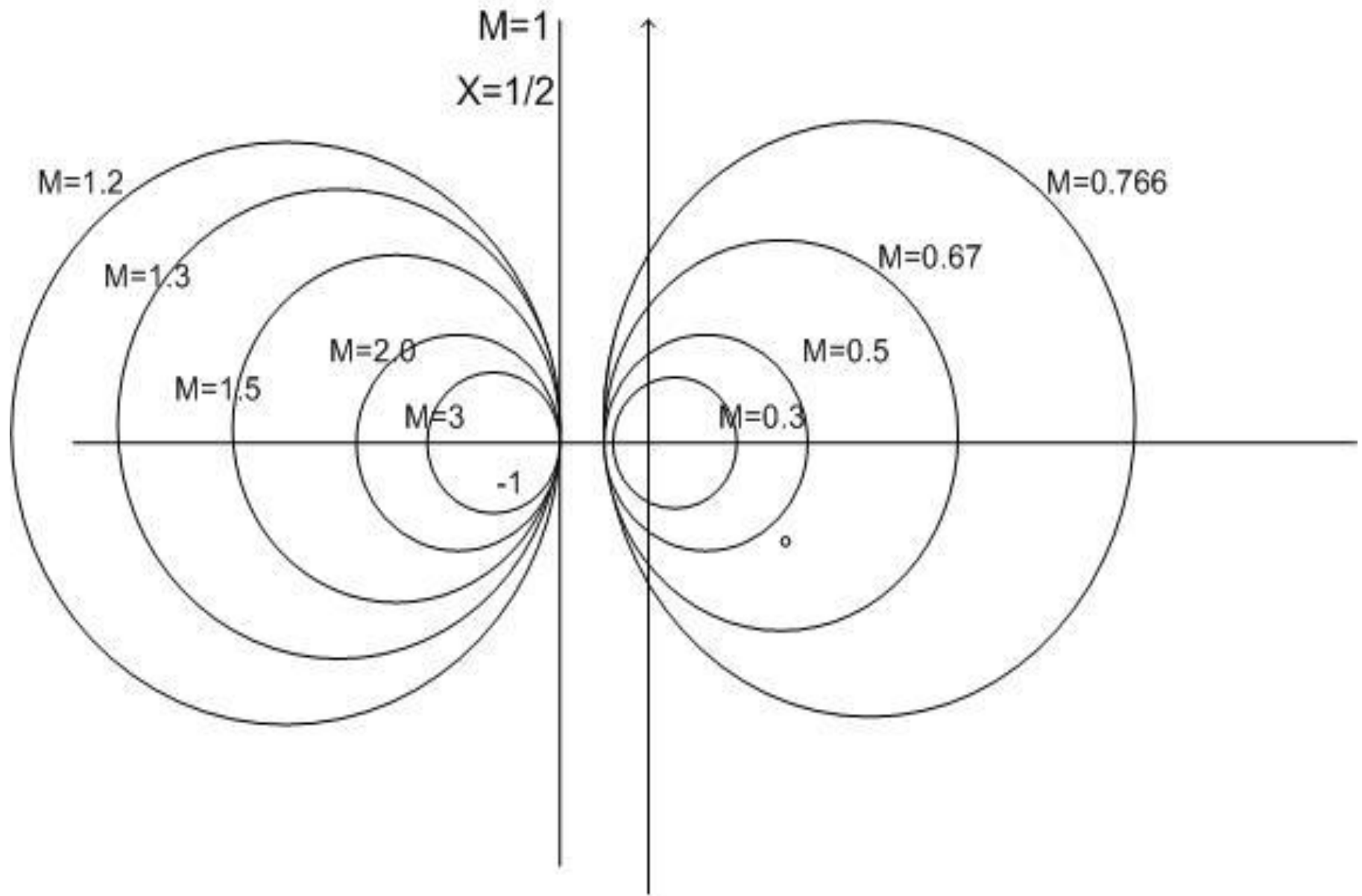


TABLE FOR CONSTRUCTION OF M-CIRCLE

S.NO.	M	CENTRE $\frac{M^2}{1-M^2}, 0$	RADIUS $\frac{M}{1-M^2}$
1.	0.3	(0.098,0)	0.329
2.	0.5	(0.33,0)	0.666
3.	0.67	(0.814,0)	1.215
4.	0.766	(1.42,0)	1.854
5.	0.833	(2.27,0)	2.72
6.	1.2	(-3.27,0)	-2.72



CONSTANT M-CIRCLES

CONSTANT N-CIRCLES (PHASE ANGLE LOCII)

From equation (1), the phase shift can be written as

$$\phi = \tan^{-1} \frac{y}{x} - \tan^{-1} \frac{y}{1+x}$$

$$\tan \phi = \frac{\frac{y}{x} - \frac{y}{1+x}}{1 + \left[\frac{y}{x} \right] \left[\frac{y}{1+x} \right]} = \frac{y}{x^2 + x + y^2}$$

let

$$\tan \phi = N$$

$$N = \frac{y}{x^2 + x + y^2}$$

$$x^2 + x + y^2 - \frac{y}{N} = 0$$

Add to $\frac{1}{4} + \frac{1}{(2N)^2}$ both sides, we get

$$(x + 1/2)^2 + \left[y - \frac{1}{2N} \right]^2 = \frac{1}{4} + \frac{1}{(2N)^2}$$

This equation represents the family of circles, with centers at $(-1/2, 1/2N)$ and radius $\sqrt{\frac{1}{4} + \frac{1}{4N^2}}$

Centre $(-1/2, 1/2N)$

$$\text{Radius} = \sqrt{\frac{N^2 + 1}{4N^2}}$$

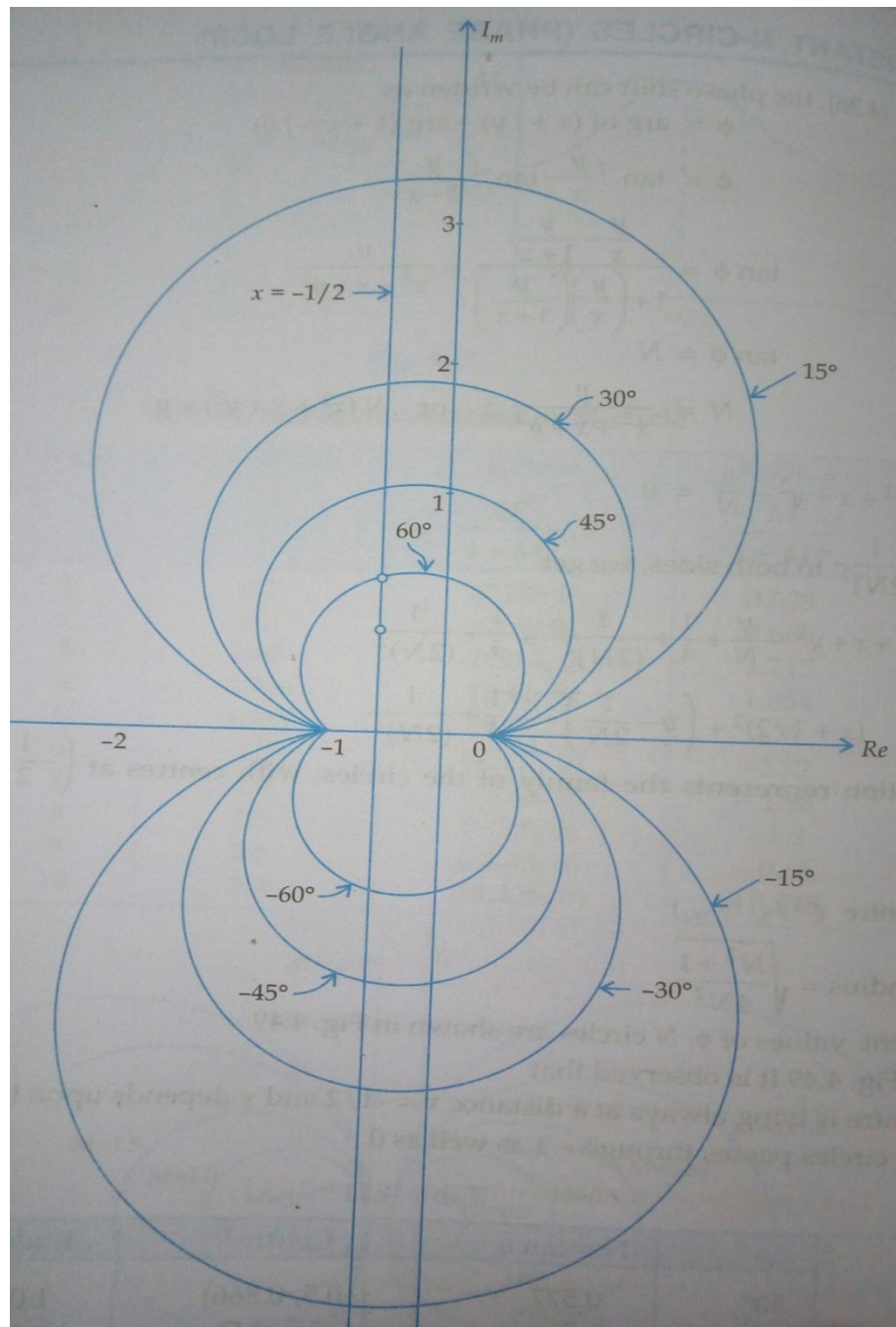
For different values of ϕ , N circles are shown in next slide

It is observed that

- (1) The centre is lying always at a distance $x=-1/2$ and y depends upon the phase shift.
- (2) All the circles passes through -1 as well as 0 .

Note: for different values of ϕ , calculate the value of N by $N=\tan\phi$, then find out centre and radius.

CONSTANT N-CIRCLES



THANK YOU